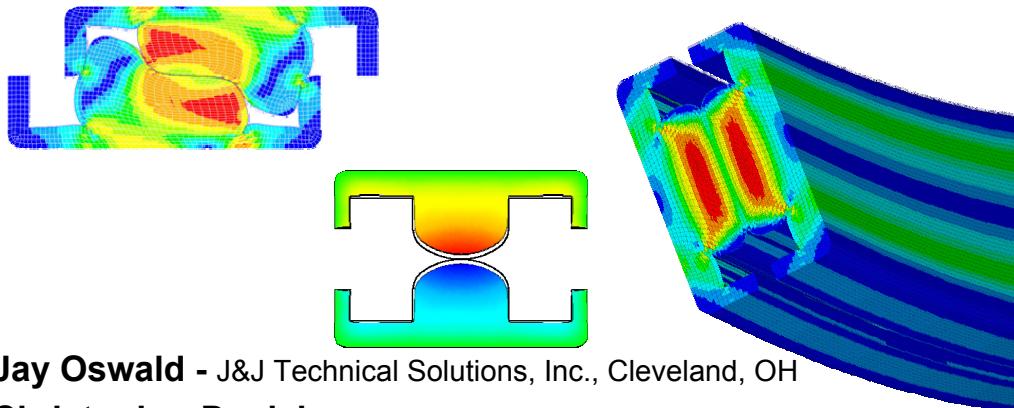


FINITE ELEMENT ANALYSIS OF ELASTOMERIC SEALS FOR LIDS

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Finite Element Analysis of Elastomeric Seals for LIDS



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Objectives & Motivation

Objective

- Create a means of evaluating seals w/o prototypes

Motivation

- Cost
 - Prototype 54" seal ~\$100k per seal pair
 - FEA license + high end workstation ~ \$30k per year
- Development time
 - 6 months lead time for a new seal design
 - Many designs per day (solution time <1 minute)
- Understanding
 - Difficult to experimentally measure strains, contact pressure profile, stresses, displacements

Part I

Hyperelastic Material Modeling

Special Properties of Hyperelastic Materials

- Fully or nearly Incompressible
 - Bulk modulus typically 100-1000x shear modulus
 - Poisson's ratio approaches 0.5
 - Problems in displacement-based FEA formulation
 - Requires B-bar or mixed u-P formulation
- Huge elastic range of deformation
 - Strains $> 80\%$ are (mostly) recoverable
 - Analysis should account for nonlinear geometry and material properties

Hyperelasticity vs. Linear Elasticity

Linear elasticity:

$$\mathbf{W} = \mathbf{C}\boldsymbol{\varepsilon}:\boldsymbol{\varepsilon}$$

(which is like: $E = \frac{1}{2} k \Delta x^2$)

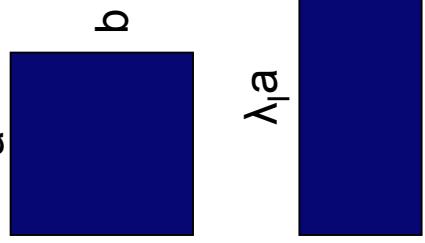
Hyperelasticity:

$$\mathbf{W} = \mathbf{f}(\mathbf{I}_1, \mathbf{I}_2, \mathbf{I}_3)$$

$$\text{or } \mathbf{W} = \mathbf{f}(\lambda_{\|}, \lambda_{\perp}, \lambda_{\text{III}})$$

$$\sigma_{ij} = \frac{\partial W}{\partial \epsilon_{ij}}$$

Definition of second Piola-Kirchoff stress from strain energy density and Green-Lagrange strain

$$\begin{aligned} I_1 &= \lambda_I^{-2} + \lambda_H^{-2} + \lambda_M^{-2} \\ I_2 &= \lambda_I^{-2} \lambda_H^{-2} + \lambda_H^{-2} \lambda_M^{-2} + \lambda_M^{-2} \lambda_I^{-2} \\ I_3 &= \lambda_I^{-2} \lambda_H^{-2} \lambda_M^{-2} = 1 + \left(\frac{\Delta V}{V} \right)^2 \end{aligned}$$


$\lambda_{\|} a$ $\lambda_{\perp} b$

$\lambda_I, \lambda_H, \lambda_M$: principal stretch ratios

I_1, I_2, I_3 : strain invariants

J : Jacobian (volume ratio)

Some forms of the work function

Polynomial models: (Mooney-Rivlin, Neo-Hookean)

$$W = \sum_{i+j=1}^N C_{ij} (\bar{I}_1 - 3)^i (\bar{I}_2 - 3)^j + \sum_{k=1}^N \frac{1}{d_k} (J - 1)^{2k}$$

Yeoh model: j=0, neglects second strain invariant

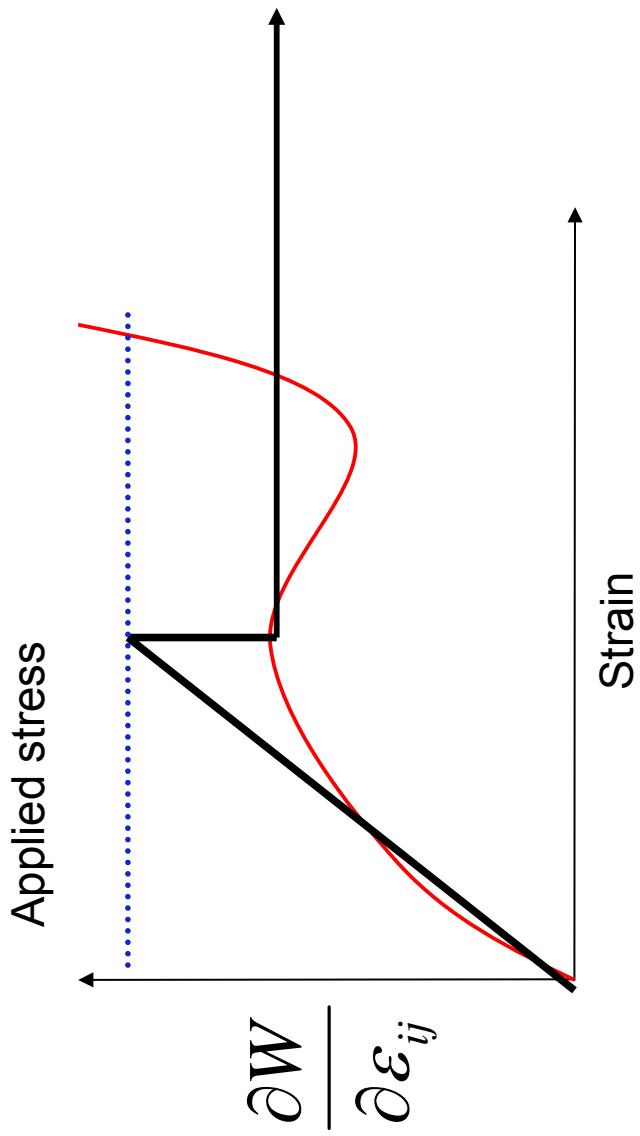
- For plane strain Yeoh is equivalent to general polynomial form because $\mathbf{I}_1 = \mathbf{I}_2$

Comparison of lowest order terms for a 50 durometer material

$$1/d_1 \approx 200,000 \quad C_{1,0} \approx 40$$

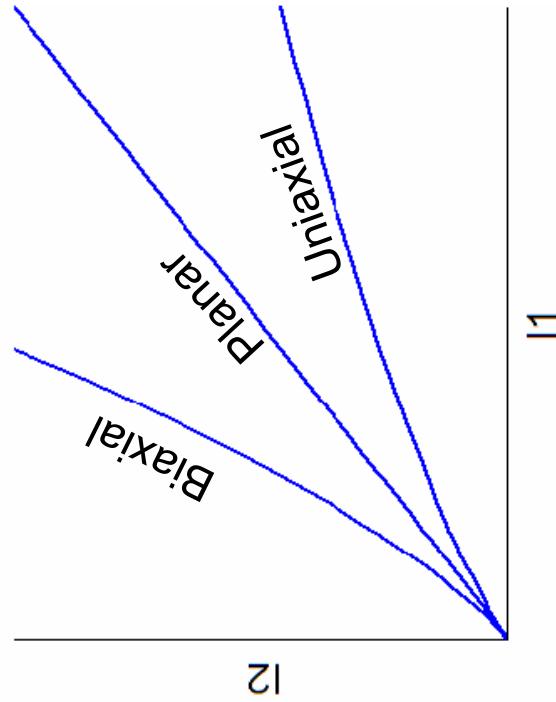
Constraints on the work function

- Zero strain must have zero energy ($W(0) = 0$)
- Zero strain must have zero stress ($W'(0) = 0$)
- Second derivative must be positive ($W''(\varepsilon) > 0$ for all ε)



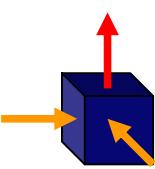
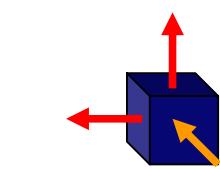
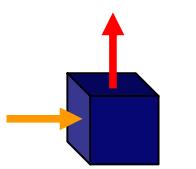
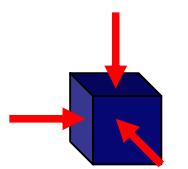
Determining W

- Fit W to experimental stress-strain states
 - Three basic strain modes
 - Uniaxial tension
 - Biaxial tension
 - Planar tension
 - All deformation falls between uniaxial and biaxial – ($|I_3| = 1 \rightarrow$ incompressible)



Energy density function of a hyperelastic material

Basic strain states of a nearly incompressible material

Load	Strain	Stretch Ratios
Uniaxial		$\lambda_I = \frac{1}{\lambda_{II}^2} = \frac{1}{\lambda_{III}^2}$
Biaxial		$\lambda_I = \lambda_{II} = \frac{1}{\sqrt{\lambda_{III}}}$
Planar		$\lambda_I = \frac{1}{\lambda_{II}}, \lambda_{III} = 1$
Volumetric		$\lambda_I = \lambda_{II} = \lambda_{III} < 1$

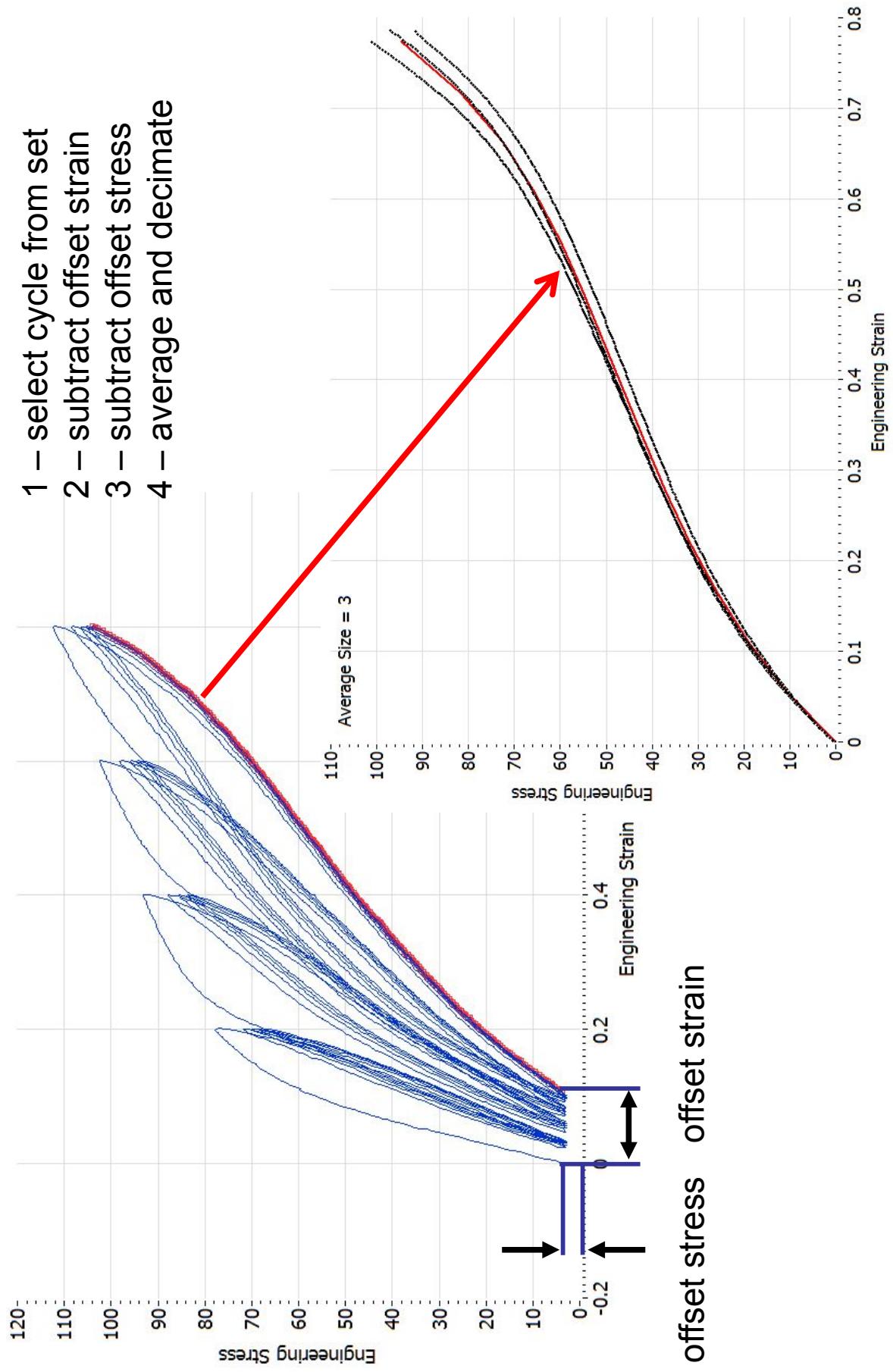
Material Tests Performed

- Materials: XELA-SA-401, S0899-50, S0383-70
 - 40, 50, 70 durometer hardness
- Test parameters
 - Various temperatures
 - -50, 23, 50, & 125 °C
 - 3 specimens per test
 - Uniaxial, planar, biaxial tension & volumetric
 - 20,40,60,80 % strain increments
- Other properties:
 - Coefficient of friction (elastomer on elastomer), thermal conductivity, heat capacity, density, emissivity, absorptivity

This data will be published soon in a NASA technical publication

Data Processing

- 1 – select cycle from set
- 2 – subtract offset strain
- 3 – subtract offset stress
- 4 – average and decimate

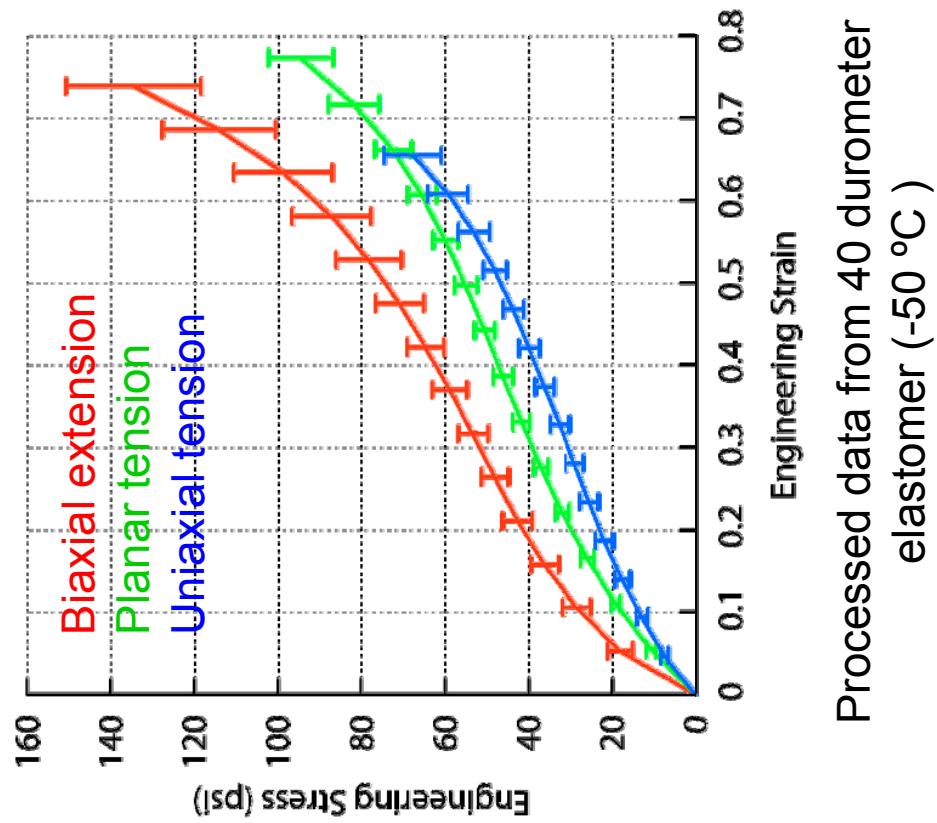


Processed Material Data

Uncertainty based off student's t distribution from multiple specimens

Results can be curve fitted to determine material property constants

This can be done as a function of temperature



Processed data from 40 durometer elastomer (-50 °C)

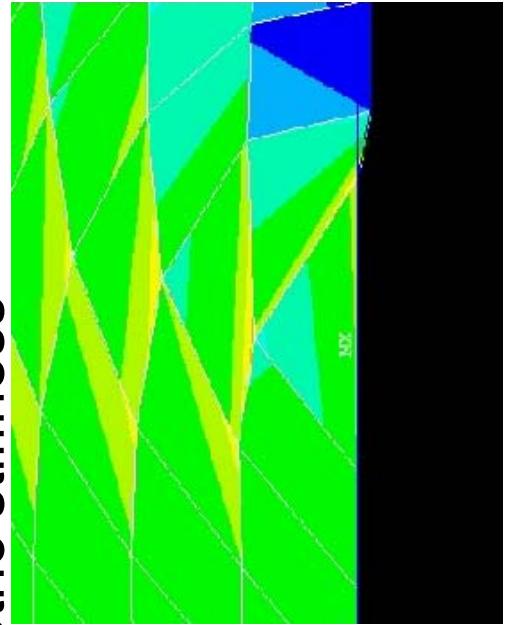
The strain energy density is the area under the curve for each deformation

Part II

Finite Element Analysis of Seals

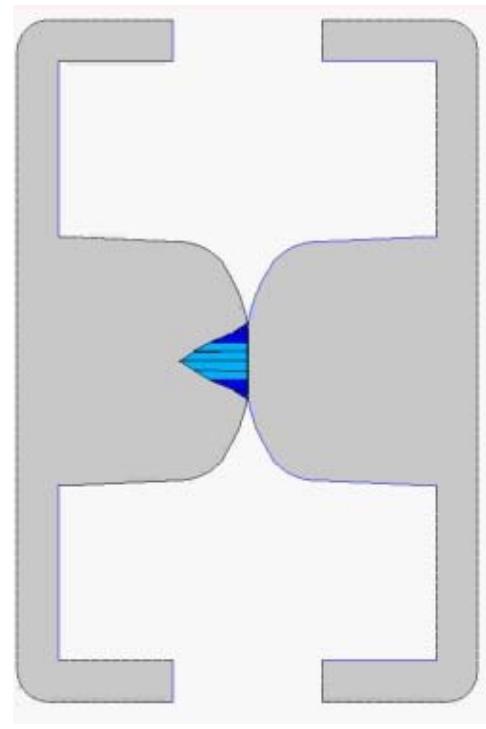
Hints for Elastomeric FEA

- 1) Stay away from triangular elements
 - Elements with 2 displacement BC will have only 1 degree of freedom due to incompressibility
- 2) Low order elements converge easiest 4-node brick works well
- 3) Sliding contact may require non-symmetric stiffness matrices for large friction coefficients
- 4) Watch corners for element distortion
- 5) u-P element formulation is most stable
- 6) Check for stability of material models

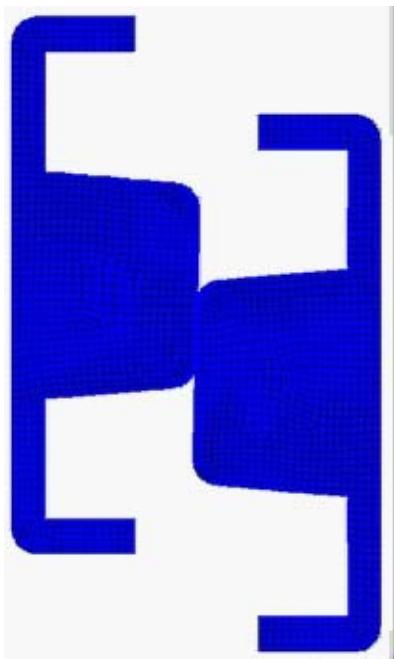


Severe element edge distortion
Analyses did not converge

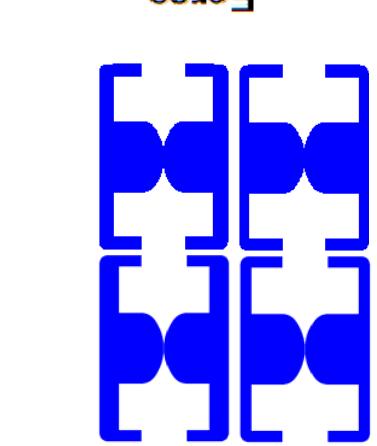
Types of FEA models of LIDS seals



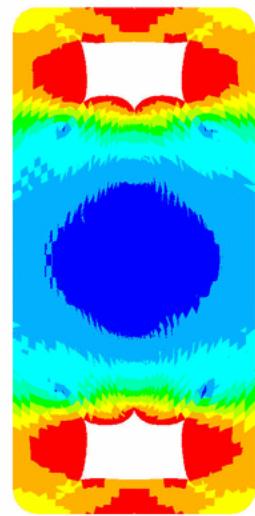
Aligned seal – contact pressure



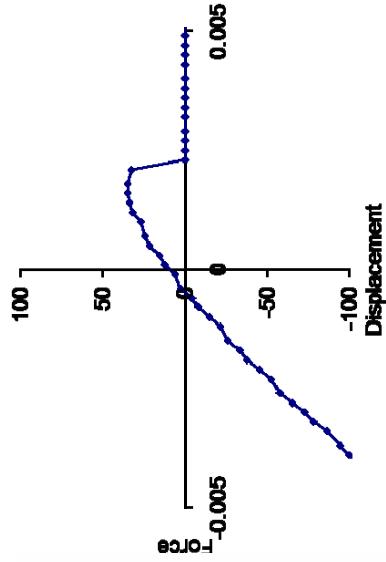
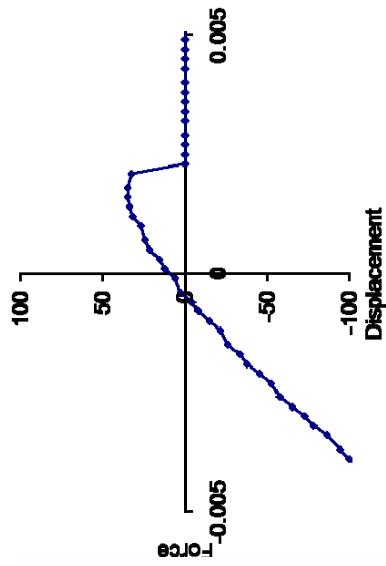
Misaligned seal
Principal strains



Aligned seal – contact pressure

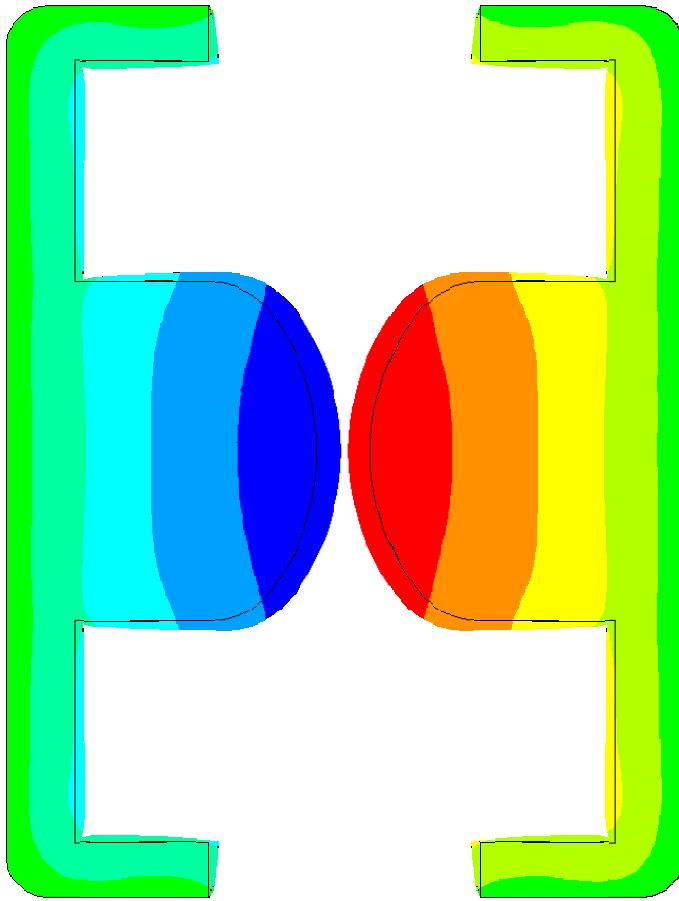


Gaskoseal adhesion analysis with
cohesive elements at contact



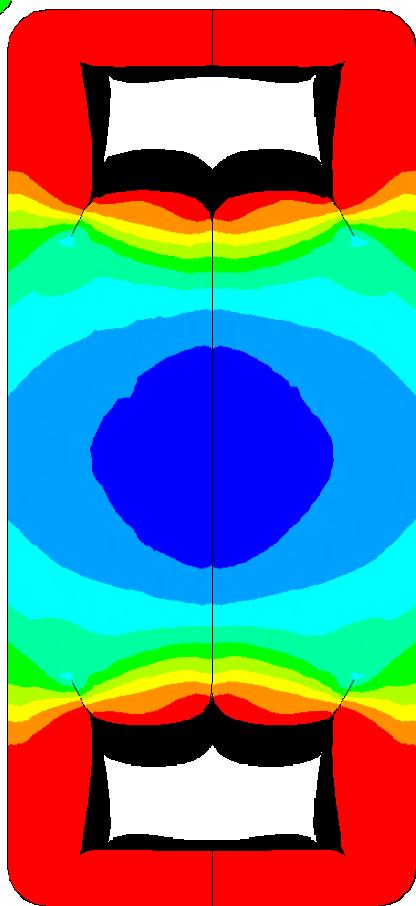
Seal Thermal Analyses

- CTE of elastomers is **very high**
 - $350 \times 10^{-6} \text{ }^{\circ}\text{C}^{-1}$
 - Al: $24 \times 10^{-6} \text{ }^{\circ}\text{C}^{-1}$



Comparison of compression at 25°C (front) and 125°C (back). Contours are axial stress.

γ displacement of seals with 100°C rise in temperature, black outline indicates original geometry



Summary

- Need 4 experimental strain states to
 - choose energy density function
 - fit material constants
 - determine compressibility of material
- Hyperelastic material present new challenges
- FEA analyses for LIDs
 - Force vs. displacement and pressure contours
 - Aligned & misaligned cases
 - Thermal expansion
 - Tolerance studies
 - Adhesion analysis

Further reading/information

- ANSYS gives excellent background for element technology/hyperelasticity
 - Nonlinear element technology
 - <http://www.ansys.com//assets/tech-papers/nonlinear-element-tech.pdf>
 - Hyperelasticity
 - http://www.tsne.co.kr/board/download.asp?strFileName=conflong_hyperel.pdf&dr=ansys
- Future publications of material properties, analysis, etc. will be posted on <http://www.grc.nasa.gov/WWW/structuralseal>